Faculty of Engineering

Philadelphia University Mechanical Vibration (620414) Mechanical Eng. Dep.

First Exam

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Student Name:

Problem #1: chose the correct answer and draw a table in the answer sheet and fill it with your choices. (6marks)

Student ID number:

1. How many degree of freedom the system shown in the figure has											
a. 0	<mark>b. 1</mark>	c.2	d.3	e. infinite							
2. Fi a. b. c. d. e.	Il the blank: the mas Store kinetic energy Store potential energy Dissipate energy a+b Has no rule at all	s in vibrating	system								
3. Fi the f	nd the equivalent sti igure where	k									
a. 10	0N/m b. 400N/m	c.300N/m	<mark>d.250N/m</mark>	e. 40N/m							
4. Find the natural frequency for single DoF system if k = 1000N/m and m=10kg.											
<mark>a. 10</mark>	rad/s b. 10 I	Hz	c.100 rad/s	d.100 Hz	e. 40N/m						
5. Which of the following represents harmonic motion. Where a, b, c, and d are constants											

a. x(t) = ax + bb. $x(t) = a \ln(x^c) + b$ c. $x(t) = a \cos(bt) + c \sin(bt)$ d. $x(t) = a \cos(bt - d)$ e. c+d

6. if single DoF system with k = 1000N/m and m=10kg has the following response: $x(t) = A\cos(\omega_n t - \phi)$. If the initial displacement was <u>2m</u> and the initial velocity equal zero. Then the values of <u>A</u>, <u> Φ </u> and ω_n will be respectively:

- a. $10m, 30^{\circ}, 10 \text{ rad/sec}$
- b. $2m, 90^{\circ}, 10 \text{ rad/sec}$
- c. 2m, 0° , 10 rad/sec
- d. $10m, 0^{\circ}, 10 \text{ rad/sec}$
- e. None of the above is correct

Q	1	2	3	4	5	6
Answer	В	Α	D	Α	Ε	С

Tuesday 09/04/2013

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Solution:

name it (θ)

deflection as follow:

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1. Equation of motion

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Problem #2: Draw the free-body diagram and derive the equation of motion using Newton's second law of motion for the systems shown in figure and find its natural frequency in terms of system parameters (6 marks)

m X(I)k (r)(0 a. when the mass is added to the system, a static deflection occurs in spring equivalent to the initial rotation. Let us name this initial rotation (θ_0). b. the motion starts from the static point and let us Т c. the force generated in spring is related to the angular $F_s = k(r)(\theta + \theta_o)$ F.B.D Т d. the governing equation for the mass (m):

m

mg

////

x(t)

$$mg - T = m\dot{x}$$
 Eq.1

e. the governing equation for the pulley (J_o) :

$$J_o \ddot{\theta} = T(4r) - k(r)(\theta + \theta_o)(r) \dots \text{Eq.2}$$

f. according to static equilibrium(when $\theta=0$): $mg(4r) = k(r\theta_o)(r) \Rightarrow \theta_o = \frac{4mg}{rk}...Eq.3$

g. substitute Eq.s 1 and 3 into Eq.2:

$$J_{o} \ddot{\theta} = \left(mg - m\ddot{x}\right)(4r) - kr^{2}\left(\theta + \frac{4mg}{rk}\right)$$
$$J_{o} \ddot{\theta} - mg(4r) + m\ddot{x}(4r) + kr^{2}\theta + (4r)mg = 0 \Longrightarrow J_{o} \ddot{\theta} + m\ddot{x}(4r) + kr^{2}\theta = 0$$

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h. x to θ relation: $x = 4r\theta \rightarrow \ddot{x} = 4r\ddot{\theta}$ then the governing equation becomes:

$$J_{o}\ddot{\theta} + m\ddot{\theta}(16r^{2}) + kr^{2}\theta = 0 \Longrightarrow (J_{o} + 16mr^{2})\ddot{\theta} + (kr^{2})\theta = 0$$
$$\omega_{n}\sqrt{\frac{(kr^{2})}{(J_{o} + 16mr^{2})}}$$

Problem #3: if the system shown in the figure represents a rod connected to linear springs at its ends and a single torsional spring at point A. (8 marks)

If the rod has the following parameters:

1. m = 4.5 kg.

2. l = 1m

$$3. \quad J_o = \frac{1}{9}ml^2$$

And the springs stiffness(s) are:

1. k = 150 N/m

2. $k_t = 50 \text{ N.m/dgree}$

Find:

1. The mathematical model that govern this system

2. Find the natural frequency of this system 3. What is the response of this system if the initial displacement equal 10° and the initial velocity equal zero.

Assume small angular deflection

Solution:

1. The coupled springs connected at the ends of the rod are connected as parallel springs. So, the equivalent stiffness for it will equal k+k=2k.

2. The F.B.D for this system is shown as after giving it an initial excitation (θ)

3. Assume the positive direction is the counterclockwise which in this case, θ is positive and the other moment parts (springs and weight) are negative because they try to rotate the system clockwise.

4. Apply Newton's 2nd law of motion to the system:





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$$\sum M_{A} = J_{o} \ddot{\theta}$$
$$J_{o} \ddot{\theta} = -2kx_{1} \left(\frac{l}{3}\right) - kx_{1} \left(\frac{l}{3}\right) - 2kx_{2} \left(\frac{2l}{3}\right) - kx_{2} \left(\frac{2l}{3}\right) - mg\left(\frac{l}{2} - \frac{l}{3}\right)\theta - k_{t}\theta$$

5. Rearrange the terms

$$J_{o}\ddot{\theta} + kx_{1}(l) + kx_{2}(2l) + mg\left(\frac{l}{6}\right)\theta + k_{t}\theta = 0$$

6. Now let us relate the translational distances $(x_1 \text{ and } x_2)$ to θ :

$$\sin(\theta) \approx \theta = \frac{x_1}{l/3} \Longrightarrow x_1 = \theta\left(\frac{l}{3}\right)$$
$$\sin(\theta) \approx \theta = \frac{x_2}{2l/3} \Longrightarrow x_2 = \theta\left(\frac{2l}{3}\right)$$

7. So the mathematical model becomes:

$$J_{o} \ddot{\theta} + k \left(\theta \frac{l}{3}\right)(l) + k \left(\theta \frac{2l}{3}\right)(2l) + mg\left(\frac{l}{6}\right)\theta + k_{t}\theta = 0$$
$$J_{o} \ddot{\theta} + \theta \left(k \frac{l^{2}}{3}\right) + \theta \left(k \frac{4l^{2}}{3}\right) + mg\left(\frac{l}{6}\right)\theta + k_{t}\theta = 0 \Longrightarrow J_{o} \ddot{\theta} + \left\{k \frac{l^{2}}{3} + k \frac{4l^{2}}{3} + \frac{mg}{6} + k_{t}\right\}\theta = 0$$

8. Substitute the physical parameters of the system in the final format of the mathematical model:

$$J_{O} = \frac{1}{9}ml^{2} = \frac{2}{9}kg.m^{2} \Rightarrow \frac{1}{2}\ddot{\theta} + \left\{150\frac{1^{2}}{3} + 150\frac{4(1)^{2}}{3} + \frac{(4.5)(9.81)}{6} + 50\right\}\theta = 0 \Rightarrow 0.5\ddot{\theta} + 307.4\theta = 0$$

9. And hence, the natural frequency equal:

$$\omega_n = \sqrt{307.4/0.5} = 24.8 \, rad/s$$

10. This system is a single DoF undamped free vibration system and the response of such system is given as:

$$\theta(t) = \theta_o \cos(\omega_n t) + \frac{\dot{\theta}_o}{\omega_n} \sin(\omega_n t) = A \cos(\omega_n t - \phi) = A_o \sin(\omega_n t + \phi)$$

1.
$$\phi = \tan^{-1}\left(\frac{\dot{\theta}_o}{\theta_o\omega_n}\right) = \tan^{-1}\left(0^o\right) = 0^o, \phi_o = \tan^{-1}\left(\frac{\theta_o\omega_n}{\dot{\theta}_o}\right) = \tan^{-1}\left(\infty\right) = \frac{\pi}{2}$$

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2.
$$A = \sqrt{\theta_o^2 + \left(\frac{\dot{\theta}_o}{\omega_n}\right)^2} = \theta_o = 10^\circ = \frac{\pi}{18}$$

So the solution can be:

1.
$$\theta(t) = \theta_o \cos(\omega_n t) + \frac{\dot{\theta}_o}{\omega_n} \sin(\omega_n t) = \frac{\pi}{18} \cos(24.8 t)$$

2.
$$\theta(t) = \frac{\pi}{18} \cos(24.8t)$$

3.
$$\theta(t) = \frac{\pi}{18} \sin\left(24.8t + \frac{\pi}{2}\right)$$

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